PATENT **SPECIFICATION**

DRAWINGS ATTACHED

1.068.610



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COMPLETE SPECIFICATION

Magnetically Actuated Fluid Flow Control Valve

We, Skinner Precision Industries, Inc., a corporation organised under the laws of the State of Connecticut, United States of America, of New Britain, Connecticut, United States of America, do hereby declare the invention, for which we pray that a patent may be granted to us, and the method by which it is to be performed, to be particularly described in and by the following 10 statement:-

This invention generally relates to fluid flow control valves and is particularly directed to the provision of an improved magnetically

operated valve. A valve according to the present invention comprises a body having a flow passageway, inlet and outlet ports opening into said pas-sageway, a valve member in said passageway movable from a first position at one end of said passageway, closing communication between said ports, to a second position at an opposite end of said passageway, establishing communication between said ports, means forming a first magnetic flux path having a portion thereof extending between said first and second positions of said valve member and including said valve member, means forming a second magnetic flux path and including a member formed of magnetic 30 material extending transversely to said first magnetic flux path at a location intermediate said first and second positions of said valve member, a permanent magnet positioned in said first magnetic flux path and producing 35 a first magnetic flux therein urging said valve member toward one of said first and second positions, and second magnet means for selectively producing a second magnetic flux in said second magnetic flux path in opposition to said first magnetic flux for shifting said valve member to the other of said first and second positions.

The invention is illustrated in the accom-

panying drawings in which:-

Figure 1 is a cross-sectional elevational 45 view of a valve; Fig. 2 is a fragmental cross-sectional view

taken generally along lines 2-2 of Fig. 1; Fig. 3 is a schematic view of an actuating circuit for the valve of Fig. 1;
Fig. 4 is a cross-sectional elevational view

of modification of the valve of Fig. 1;

Fig. 5 is a cross-sectional elevational view of another modification of the valve of Fig.

Fig. 6 is a diagrammatic view of the valve illustrating a magnetic flux arrangement employed for holding the valve member in its

flow-controlling positions;
Fig. 7 is a diagrammatic view similar to Fig. 6 but illustrating the magnetic flux arrangement employed for shifting the valve into one of its flow-controlling positions;

Fig. 8 is a view similar to Fig. 7 but showing the flux arrangement for shifting the valve into the opposite flow-controlling posi-

Fig. 9 is a cross-sectional elevational view of another modification of the valve of Fig 1; Fig. 10 is a cross-sectional elevational view of another embodiment;

Fig. 11 is a plan view of a valve representing still another embodiment;

Fig. 12 is a cross-sectional view taken generally along lines 12-12 of Fig. 11; Fig. 13 is a schematic view of an alternative

actuating circuit for the valve; Fig. 14 is a cross-sectional elevational view of a valve representing still another embodi-

ment: 15 is a cross-sectional view taken Fig.

generally along lines 15-15 of Fig. 14; and Fig. 16 is a cross-sectional view taken generally along lines 16—16 of Fig. 15.

Referring to the drawings in detail, Fig. 1 shows a valve including a body, generally designated 2, formed from a suitable nonmagnetic material and having a flow passage

4 extending therethrough. In the opposite ends of the flow passage 4 there are secured screw plugs 6 and 8 formed from a suitable The porting in the valve magnetic material. of Fig. 1 is shown for illustrative purposes only and includes first and second passages 7 and 9 provided in the screw plugs 6 and 8 and opening into the flow passage 4 and a third passage 10 provided in the body 2 in communication with the flow passage 4 intermediate the ends thereof.

Flow between the passages 7, 9 and 10 is controlled by a valve plunger, generally designated 12, slidably received in the flow passage 4 for movement between two positions at the opposite ends thereof as determined

In one position by the screw plugs 6 and 8. (shown in Fig. 1) of the valve plunger 12, communication between the passages 9 and 10 is established while the passage 7 is closed and in the opposite position of the valve plunger 12, communication between the passages 7 and 10 is established while passage 9 is closed; the inner faces of the screw plugs 6 and 8 being provided with seats 13 for the valve plunger 12, in the illustrated embodiment. The valve plunger 12 is actuated

into these opposite flow-controlling positions and is releasably held therein solely by mag-

netic forces. The valve plunger 12 is provided with a permanent magnet core or centre piece 14 formed from any suitable material such as "Alnico VIII" and having a uniform cross section shown as circular in Fig. 2. In the preferred embodiment, on the opposite ends of the core 14 there is fixed, such as by brazing, a pair of identical pole pieces 16 formed from a suitable magnetic material. The pole pieces 16 are dimensioned to slidably engage the walls of the flow passage 4 to

plugs 6 and 8 to thereby close the passages 7 and 9. Longitudinal grooves 17 are provided in the periphery of the pole pieces so as to convey fluid between passages 7, 9 and 10. Additionally. for improved sealing results, elastomeric seal inserts 18 may be provided in the pole pieces

thereby guide the plunger 12 and further to

seat against the inner faces of the screw

16 so as to be co-operable with the seats 13

on the screw plugs 6, 8.

In accordance with another aspect of the present invention, there is provided an electromagnet 20 including a core 21 of E-shaped cross-section mounted on the body 2 to one side of the flow passage 4 with the legs 22, 23 and 24 of the core 21 extending generally normal to the axis of the flow passage 4; the outer legs 22, 23 straddling the ends of the body 2 while contacting the plugs 6, 8; and the central leg 24 extending into the body 2 midway between the plugs 6, 8 and terminating adjacent the pole pieces 16 of the valve plunger 12.

In the illustrated embodiment, the electromagnetic core 21 is formed by a yoke piece of suitable magnetic material which provides the outer legs 22, 23; and a steel screw which provides the central leg 24 while also serving to secure the electromagnet 20 to the body The electromagnet 20 further includes a wire or coil 26 wound on the central leg 24 of the core above the valve body 2 where it is isolated and may be conveniently removed and replaced. The O-ring seals 28 are preferably provided around the plugs 6, 8 so as to prevent leakage as well as to seal off the coil

26 from the flow media. An enclosure cap may be provided to cover the electromagnet and referring to Fig. 4, a commercial embodiment of the invention is illustrated wherein the outer legs 35 of the core of the electromagnet are provided by the side walls of a cylindrical enclosure cap 37 thus eliminating the need for a separate yoke piece for the electro-magnetic core, as in the above-described embodiment. In the valve of Fig. 4 the outer core legs 35 are connected to the plugs 47 to thereby provide a low reluctance magnetic flux path by means of an apertured base plate 57 which is formed of magnetic material and is received on the valve body around an external flange 58. The base plate 57 has depending from its ends, a pair of ears 59 which are received in grooves 65 provided in the plugs 47 to thereby establish contact with the plugs 47 as well as to he'd the same in position in the valve body. The enclosure cap 37 is secured to the valve body on the base plate 57 by a threaded stud 95 and a co-operable nut 96, the screw 95 further providing the central leg of the elec-

tromagnet core. Referring to Fig. 6 in conjunction with the valve of Fig. 1, it will be seen that the permanent magnet 14 produces a flux 29 that extends along the path from the north pole of the permanent magnet to the screw plug 6 through the outer core leg 22 and back to the south pole of the permanent magnet through the central core leg 24 as well as through the outer core leg 23 and screw plug 8. The flux 29 of the permanent magnet additionally extends from the north pole directly to the central core leg 24 and then directly back to the south pole of the permanent magnet. Since the total of the gaps W and Y is less than that of gaps X and Z, manent magnet. the flux densities and thus the attractive forces across gaps W and Y will be greater than that across gaps X and Z. The result will be that the valve plunger 12 will be held against the screw plug 6 under the attractive forces across gaps W and Y.

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When it is desired to shift the valve plunger 12 to the opposite flow-controlling position, the electro-magent is pulse-energized with a proper polarity so as to induce a flux 31 (see Fig. 7) that will reinforce the permanent 130 1,068,610

magnet flux 29 across gaps X and Z while opposing the permanent magnet flux 29 across gaps W and Y as denoted by the arrows The result will be that the valve in Fig. 7. plunger 12 will shift to the right (as viewed in the drawings) into the opposite flow-con-trolling position since the density and therefore the attractive forces across gaps X and Z will be greater than those across gaps W and Y.

In order to maintain the valve plunger 12 in the right-hand flow-controlling position, it is unnecessary to maintain the electromagnet energized since the flux of the permanent magnet across gaps X and Z will become effective to hold the valve plunger 12 against the screw plug 8 upon de-ener-

gization of the electromagnet.

Referring to Fig. 8, in order to shift the valve plunger back to the left-hand flow-controlling position the electromagnet is pulseenergized with a polarity opposite to that described above. As illustrated by the arrows in Fig. 8, this will induce a flux 33 that will reinforce the permanent magnet flux 29 across gaps W and Y while opposing the flux 29 across gaps X and Z and thus the valve plunger 12 will shift to the left-hand flow-controlling position against the screw plug 6 under the attractive forces across gaps W and Y. This shifting of the valve plunger 12, as well as that described above, is not only caused by the net attractive forces produced by the combined flux of the permanent magnet and the electromagnet, but is additionally caused by repulsion forces which are produced by the interaction between the flux of the permanent magnet and the electromagnet.

Referring to Fig. 3, there is shown an illustrative electrical circuit 30 for the electromagnet 20, the circuit 30 including a battery source 32, a capacitor 34 in series with the coil 26 of the electromagnet 20 and a double throw switch 36. The capacitor 34, as included in the circuit 30, provides a simple and efficient method for reversing the energizing current of coil 26 as well as limiting the energizing current to an impulse sufficient

for shifting the valve plunger 12.

The electrical actuating circuit with its components may be conveniently provided as a compact power pack, such as illustrated in the modification of the valve shown in Fig. 5, wherein two batteries 32 (only one shown) and three capacitors 34 (only two shown) are utilized. In the valve of Fig. 5, the batteries 32 and capacitors 34 are held in place in an enclosure cap 39 by means of a recessed holder 73. The holder 73 is mounted on the electromagnet 99 by a screw 98 extending through the cap 39 and received in the central leg 97 of the electromagnet core; the screw 98 also providing a portion of the central leg of the electromagnet core.

in the modification of Fig. 4, the outer legs of the electromagnet core in the modification of Fig. 5, are provided by the enclosure cap 39.

Other actuating circuits may be employed and referring to Fig. 13, an alternative circuit 110 is illustratively shown as including an a.c. power source 112 and a pair of opposed parallel rectifiers 114 and 115 arranged to be selectively connected into the circuit by a double throw switch 116 to thereby obtain the desired current direction through the electromagnet coil 26 for shifting the valve The valve of the present invention as so utilized with a.c. power source, elimin-ates the need for "shading" devices typically required in similarly powered conventional solenoid valves.

Thus it will be apparent that the valve construction of the present invention provides for efficient actuation of the valve plunger with a minimum of power since a low reluctance flux path is provided particularly by the close positioning between the pole pieces 16 and the central leg 24 of the electromagnet 20 together with the high permeability of the pole pieces 16. Moreover, the reluctance of the electromagnet circuit is substantially constant regardless of the position of the valve plunger 12, primarily because the total of the gaps W, X, Y and Z (See Figs. 6—8) is constant for any position of the valve plunger. Additionally, the interposition of the pole pieces 16 not only serves to protect the permanent magnet 14 from de-magnetizing fields but further provides for a greater axial component of driving force to be exerted on the valve plunger 12 for shifting the same. Thus, with these features the present invention not only provides an extremely low power requirement for actua-tion, but further provides a smooth and quick acting balanced valve plunger that will seat with a minimum of impact but will nevertheless be firmly held in either of its flow controlling positions.

Referring to Fig. 9, the modified valve shown therein is generally the same as that of Fig. 1 except that the central leg 40 of the electromagnet core 42 extends across the valve plunger 44 and is apertured at 46 so as to accommodate the valve plunger 44. This positioning of the portions 48 of the central electromagnet core leg 40 closer to the pole pieces 45 of the valve plunger 44 and to extend around the valve plunger 44, not only increases the axial component of force for shifting the valve plunger 44 but further minimizes side or radially directed forces on the plunger 44 by distributing the central leg flux completely around the periphery of the

valve plunger 44.

In the modification of Fig. 9 the electromagnet core 42 is mounted to the valve body 50 through means of the outer core legs

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41 and the screw plugs 52, 54 which extend through apertures 43 in the outer core legs 41 to clamp the same against the ends of the valve body 50.

Additionally, the valve of Fig. 9 illustrates

another porting arrangement including inlet and outlet passages 53, 55 which are inter-communicated when the valve plunger 44 is in the right-hand position (shown in Fig. 9) and closed from each other when the valve plunger 44 is in the opposite left-hand posi-This flow-control arrangement is effected by providing the right-hand pole piece 45 of the valve plunger 44 with a groove 56 that will be in registry with the passage 55 when the valve plunger 44 is in the position shown in Fig. 9 to thereby establish communication between the passages 53 and Of course when the valve plunger 44 is in the left-hand position the passage 53 will be closed by the pole piece 45.

Another embodiment of the invention is shown in Fig. 10 which differs from that of Fig. 1 in that the valve plunger 60 is formed 25 from magnetic material and permanent magnets 62 are externally provided in the outer legs 64 of the electromagnetic core 63.

With the embodiment of Fig. 10 the same end results are obtained in that when the electromagnet 61 is de-energized the flux of the permanent magnets 62 will become effective to hold the valve plunger 60 against one of the screw plugs 66, 67. When it is desired to shift the valve plunger 60 to the opposite position the electromagnet coil 68 is energized by an electrical impulse of proper polarity thereby inducing a second magnetic flux which will interact with the permanent magnetic flux to produce a net force of attraction that will shift the valve plunger 60 to the opposite position as limited by one of the plugs 66, 67. Of course when it is desired to shift the valve plunger 60 back to the initial position, the coil 68 is energized with an oppositely directed current impulse.

It will be seen that in the embodiment of Fig. 10 the stationary positioning of the permanent magnets 62 out of the flow passage 69 and sealed off therefrom protects the permanent magnets 62 from demagnetization as may be caused by impact or the harmful effects of certain flow media while at the same time providing convenient access to the permanent magnets 62 for removal and replacement. Additionally, this external placement of the permanent magnets 62 permits them to be of a greater size and therefore strength and to be employed substantially

as cast.

The present invention may also be advantageously applied to provide an improved bistable valve that may be actuated manually with little effort and without the problems attendant to conventional mechanical actu-

ators. Referring to Figs. 11 and 12, another such embodiment of the invention is shown which differs from that of Fig. 1 in that it includes a manually operable permanent magnet 70 for shifting the valve plunger 72 to its opposite flow-controlling positions. The permanent magnet 70 which is shown as circular and having a central mounting aperture 71, is mounted for rotation on the valve body 74, externally thereof midway between the screw plugs 76, 77.

In the specific embodiment of Figs. 11 and 12 the mounting of the permanent magnet 70 is effected through means of a stud 78 fixed into the body 74 midway between the plugs 76, 77 and adjacent the valve plunger 72, the stud being formed from magnetic material so as to provide a path of low reluctance for the flux of the permanent magnet 70. The stud 78 has an intermediate cylindrical portion 80 received through the mounting aperture 71 of the permanent magnet 70 and further, an upper flange 82 providing a stop preventing removal of the permanent magnet 70 from the stud 78.

In the instant embodiment the valve plunger 72 is held by magnetic attraction and repulsion in either of its two flow-controlling positions against one of the plugs 76, 77, depending on the angular position of the external permanent magnet 70. When it is desired to shift the valve plunger 72, say to the right as shown in Fig. 12, the external permanent magnet 70 is rotated 90° in either the clockwise or counter-clockwise direction 100 so as to reverse the polarity effect of its magnetic flux and this, together with the flux of the internal permanent magnet 86, will be effective to produce net attractive and repulsive forces that will move the valve plunger 72 to the right-hand position (as viewed in Fig. 12) and maintain it therein.

In order to insure that the external permanent magnet 70 will be maintained in its selected positions, a magnetic detent is provided by four latch members 90, 92 formed from magnetic material and angularly spaced 90°C from each other around the periphery of the permanent magnet 70 as shown in Fig.

In the specific embodiment, the magnetic latch members 90 are separately formed each with a stepped shape and are respectively secured to the opposite ends of the valve body 74 by the plugs 76, 77 portions of which extend through an aperture in the base 91 of the latch members 90. The other magof the latch members 90. netic latch members 92 in the specific embodiment are formed by one generally U-shaped piece 93 secured to the top of the body 74 through means of the stud 78 which extends through an aperture in the latch member 93. A washer 94 formed of nonmagnetic material is interposed on the stud

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78 between the permanent magnet 70 and the latch member 93 as best shown in Fig. 12.

Thus, when the external permanent magnet 70 is moved to a position for shifting the valve 72, a magnetic force of attraction between the poles of the external permanent magnet 70 and the latch members 90, 92 will be effective to maintain the external magnet 70 in the selected position against inadvertent movement. A suitable indicator (not shown) may be provided on the top surface of the permanent magnet 70 so as to facilitate the setting thereof for shifting the valve plunger 72 into the desired flow con-

15 trolling position.

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The present invention may easily be incorporated with great advantage into multiway valve units of the type that include a plurality of separate valve members. In this regard Figs. 14, 15 and 16 show a four-way valve representing another embodiment of the present invention basically similar to the embodiment in Fig. 1 except that two flow passages 120, 121 are provided in the valve body 119 in side-by-side relationship with each passage having a separate valve plunger 124 and 125, respectively. In the shown embodiment the valve plunger 124 is adapted to control flow between the three ports 126, 127 and 128 opening into the passage 120 while the other plunger 125 is adapted to control flow between the ports 129, 130 and 131 opening into the other passage 121.

Other port and passage arrangement may be employed such as that included in a fourway reversing valve for example. As shown in Figs. 14-16, all porting is adapted to be formed in the valve body, and variations such as normally open and normally closed operations can be achieved by proper assembly of the armatures with respect to their polarity; thus, the armatures may provide simultaneous sealing of the orifices at either end of the valve or alternatively, respectively seal the orifices at opposite ends of the valve. Convenient access to the valve plungers 124, 125 for reversing their position to provide a desired valving function or for repair and replacement purposes is provided by the magnetic plugs 134, 135 which are removably held in the opposite ends of the flow passages 120, 121 by the outer legs 140, 141 of the electromagnet 136.

The electromagnet 136 as in the above-described embodiment of Fig. 1, includes an E-shaped core 137 and a single coil 138 located around the central leg 139 of the core 137; the central leg 139 being provided by a screw in the specific embodiment. However, the central core leg 139 in the embodiment of Figs. 14—16 is positioned midway between the flow passages 120 and 121, as

shown in Fig. 14.

Thus, it will be readily appreciated that

the present invention, as illustrated in the embodiment of Figs. 14—16, provides a compact multiway valve unit of increased versatility and performance requiring only a minimum of operating power despite the plurality of valve members which are employed.

From the foregoing it will be seen that the present invention provides an improved valve construction for magnetically operated valves wherein all the porting is conveniently located in the valve body so as to be effectively isolated from the actuator components, particularly the electromagnet coil in the solenoid valve embodiments. The present invention also provides this feature in a valve of extremely compact construction including a minimum of parts that may be economically manufactured and quickly and easily assembled. It will additionally be seen that the present invention provides a solenoid valve requiring an extremely low energy input for the actuation thereof which may be effected through means of a single coil while at the same time providing a smoothly operable and unattached valve member which will be automatically held in its flow controlling positions upon de-energization of the solenoid.

WHAT WE CLAIM IS:-

1. A valve comprising a body having a flow passageway and inlet and outlet ports opening into said passageway, a valve member in said passageway movable from a first position at one end of said passageway, closing communication between said ports, to a second position at an opposite end of said passageway, establishing communication between said ports, means, forming a first magnetic flux path, having a portion thereof extending between said first and second positions of said valve member and including said valve member, means forming a second magnetic flux path and including a member formed of magnetic material extending transversely to said first magnetic flux path at a location 110 intermediate said first and second positions of said valve member, a permanent magnet positioned in said first magnetic flux path and producing a first magnetic flux therein urging said valve member toward one of said first and second positions, and second magnet means for selectively producing a second magnetic flux in said second magnetic flux path in opposition to said first magnetic flux for shifting said valve member to the 120 other of said first and second positions.

2. A valve as claimed in claim 1 in which an electromagnet is provided for selectively producing the second flux, the electro-magnet core having a portion forming said member of magnetic material extending transversely to said first magnetic flux path at a location intermediate said first and second positions

of said valve member.

3. A valve as claimed in claim 2 wherein said portion of said electro-magnet core extends into the body at a point adjacent the valve member and midway between the ends of the body for providing a low reluctance path for the magnetic flux of said electro-magnet

4. A valve as claimed in claim 3 in which the electro-magnet is mounted on the body and includes a core of E-shaped cross-section having a central leg constituting said portion of the electro-magnetic core and a pair of outer legs extending generally transverse to the axis of the flow passageway and a coil positioned around the central leg of the core to one side of the flow passageway, such than an electrical impulse, supplied to the coil, will induce the second magnetic flux which, together with the first magnetic flux, will cause attraction and repulsion of the valve member out of one of said positions and into the other of said positions, depending on the polarity of energization of the coil.

5. A valve as claimed in any of the preceding claims in which portions of the body at the opposite ends of the flow passageway are formed from magnetic material, the permanent magnet providing the first magnetic flux for holding the valve member in either of said positions against said body portions.

6. A valve as claimed in claim 4 or 5 wherein the central leg of the electro-magnet

core extends across the flow passageway and has an aperture receiving the valve plunger.

7. The valve as claimed in claim 4 or 5, wherein the central leg of the electromagnet core is a screw secured in the body for holding the electro-magnet thereon.

8. A valve as claimed in claim 1, in which detent means, formed of magnetic material, is positioned adjacent said second magnet means for releasably holding the same in the selected position.

9. A valve as claimed in any of claims 4 to 7 comprising an actuating circuit for the electro-magnet including a capacitor connected in series with the electro-magnet coil and a switch having a first position wherein the capacitor is charged and a current impulse is passed through the electro-magnet coil in one direction and a second position wherein the capacitor is discharged and a current impulse is passed through the coil in the opposite direction.

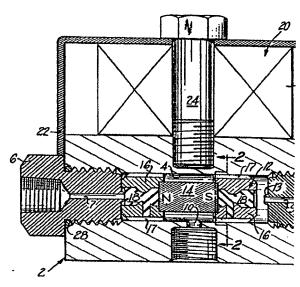
10. Valves constructed, arranged and adapted to operate substantially as hereinbefore described and as illustrated in the accompanying drawings.

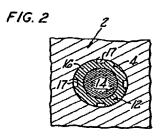
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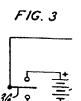
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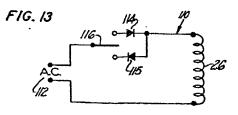
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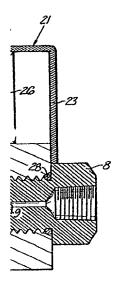
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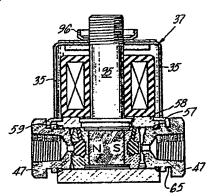
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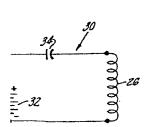
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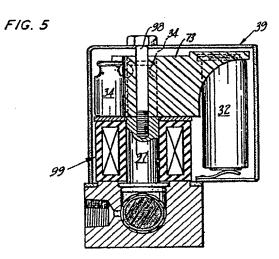
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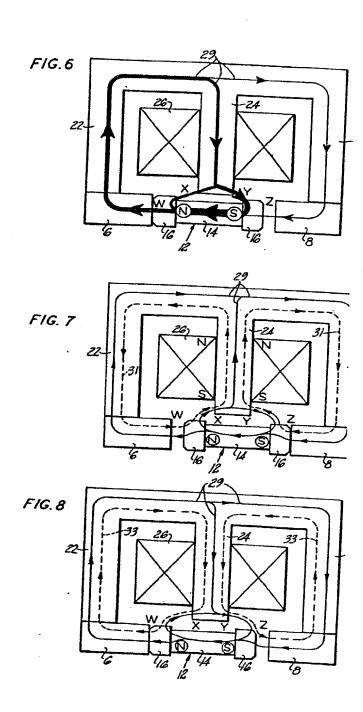








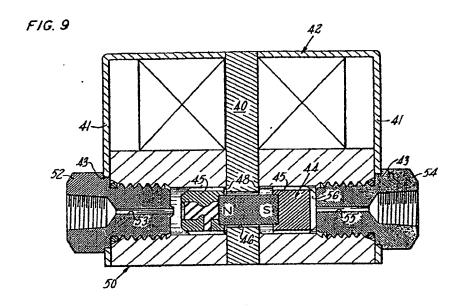
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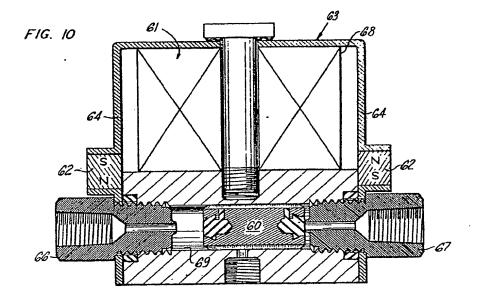


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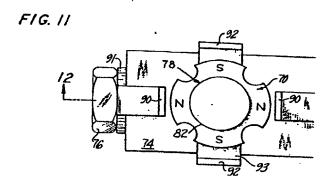
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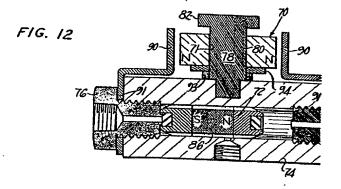




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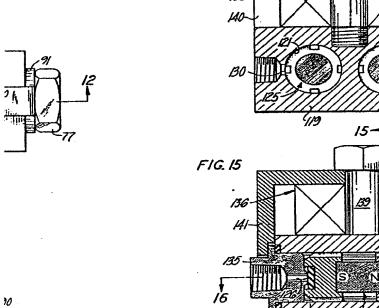
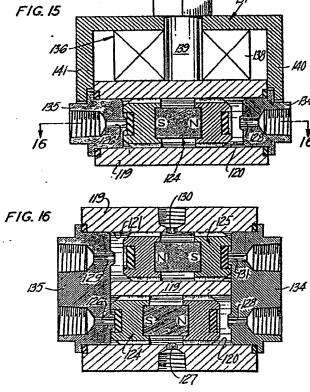


FIG. 14



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Sheets 5 & 6 FIG 15 mm FIG. 14 F1G.16 F/G. 11 F1G. 12